

10.1. a)

Bernoulli equation: $\frac{d}{dt} \left(\rho + \frac{1}{2} \rho v^2 + \rho g h \right) = 0$ (and)

ρ is pressure due to gravity, so $\rho = \rho g h$

$\Rightarrow \frac{d}{dt} \left(\frac{1}{2} \rho v^2 + 2 \rho g h \right) = 0$

$$\frac{1}{2}, 2, \rho \text{ and } g \text{ are constant w.r.t. time.}$$

v and h are functions of time.

$\Rightarrow \frac{1}{2} \cdot (v \cdot \dot{v} + \dot{v} \cdot v) + 2 \cdot g \cdot \dot{h} = 0$

$$\Rightarrow v \cdot \dot{v} + 2g \dot{h} = 0$$

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Change of volume: $\dot{V} = \pi r^2 \cdot \dot{v} = \pi R^2 \cdot \dot{h}$

$$\Rightarrow \dot{h} = \frac{r^2}{R^2} \cdot \dot{v}$$

$$\Rightarrow v \cdot \dot{v} + 2g \cdot \frac{r^2}{R^2} \cdot \dot{v} = 0$$

$$\Rightarrow \text{either } v = 0 \text{ or } \dot{v} = -2g \frac{r^2}{R^2}$$

$$\Rightarrow v(t) = \int -2g \frac{r^2}{R^2} dt = -2g \frac{r^2}{R^2} \cdot t + C$$

$$b) V(h) = V_0 - \int_0^{t_0} \dot{V}(t) dt$$

$$= \pi R^2 h_0 - \int_0^{t_0} \pi r^2 v dt = \pi R^2 h_0 - \int_0^{t_0} \frac{r^4}{R^2} dt$$

$$V(t_{\text{stop}}) = 0 \Rightarrow \int_0^{t_{\text{stop}}} \frac{r^4}{R^2} dt = \pi R^2 h_0$$

$$\Rightarrow t_{\text{stop}} = \frac{R^2}{r^2} \sqrt{\frac{\pi \cdot h_0}{g}}$$

correct answer is $\frac{R^2}{r^2} \sqrt{\frac{2h_0}{g}}$

c) Considering a single point of mass flowing out of the hole
For it holds true:

$$z(t) = z_0 + v_0 t + \frac{1}{2} g t^2 \quad \text{with the } z \text{ axis pointing downwards.}$$

$$v_0 = v, \quad z_0 = 0 \Rightarrow z(t) = vt + \frac{1}{2} g t^2$$

$$\Rightarrow \frac{1}{2} g t^2 + vt - y = 0 \Rightarrow t^2 + \frac{2v}{g} t - \frac{2y}{g} = 0 \Rightarrow t = \sqrt{\frac{v^2}{g^2} + \frac{2y}{g}} - \frac{v}{g}$$

Now considering the equation of continuity:

$$A_{hole} \cdot v = A_{jet} \cdot v_j$$

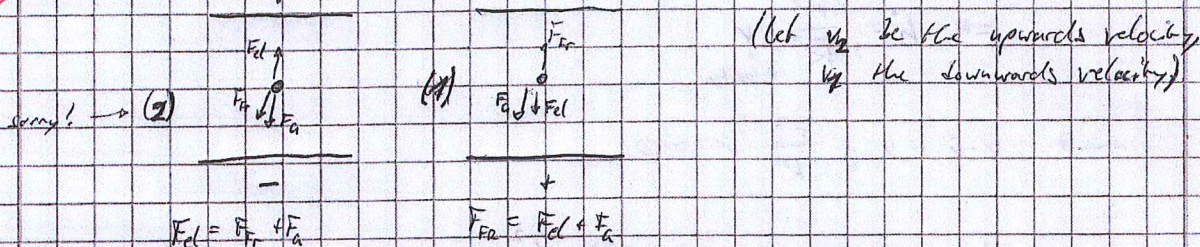
$$v_j = (v + gA) = \left(v + g \cdot \left(\sqrt{\frac{v^2}{g^2} + \frac{2V}{g}} - \frac{v}{g} \right) \right)$$

$$= v + \left(\sqrt{v^2 + 2yg} - v \right) = \sqrt{v^2 + 2yg}$$

$$\Rightarrow \pi r^2 \cdot v = \pi r_j^2 \cdot \sqrt{v^2 + 2yg}$$

$$\Rightarrow r_j^2 = \frac{r^2 v}{\sqrt{v^2 + 2yg}} \quad \Rightarrow r_j = r \cdot \frac{\sqrt{v}}{\sqrt{v^2 + 2yg}} \quad (+4)$$

(+10) Q.2 a) Velocities are constant due to a friction force proportional to velocity.



b) Electrical force:
 $F_{el} = qE$

Gravity: $F_g = m \cdot g = V \cdot \rho \cdot g = 4\pi r^3 \rho g$ (Technically, a buoyancy force of $4\pi r^3 \rho_{air} g$ has to be considered, but $\rho_{air} \ll \rho_{oil}$, so this can be neglected in our terms of accuracy)

Friction: $F_{fr} = 6\pi \eta r v$

(1) ~~$\frac{4}{3} \pi r^3 \rho g + qE = 6\pi \eta r v_1$~~

(2) ~~$\frac{4}{3} \pi r^3 \rho g - qE = 6\pi \eta r v_2$~~

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$$\frac{4}{3} \pi r^3 \rho g + 6\pi \eta r v_2 = qE \Leftrightarrow \frac{4}{3} \pi r^3 \rho g - qE = -6\pi \eta r v_2$$

(1) + (2) yields $\frac{8}{3} \pi r^3 \rho g = 6\pi \eta r (v_1 - v_2)$

$$\Rightarrow r^2 = \frac{9}{4} \frac{\eta (v_1 - v_2)}{\rho g} \Rightarrow r = \frac{3}{2} \sqrt{\frac{\eta (v_1 - v_2)}{\rho g}} \quad (+)$$

(1) - (2) yields: $2qE = 6\pi \eta r (v_1 + v_2)$

$$\Rightarrow q = \frac{3\pi \eta (v_1 + v_2)}{E} \cdot r$$

Plugging in r : $q = \frac{3\pi \eta (v_1 + v_2)}{E} \cdot \frac{3}{2} \sqrt{\frac{\eta (v_1 - v_2)}{\rho g}}$

$$= \frac{9\pi}{2E} \sqrt{\frac{\eta^3 (v_1 - v_2)^3}{\rho g}} \cdot (v_1 + v_2) \quad (+)$$

c) Plugging in values:

(Wolfram Alpha for calculation) $\Rightarrow g \approx 0,48 \cdot 10^{-10} \text{ C}$

$\frac{\varphi}{e} = 3$ (+)

10.3 a) $M_{\max} = \frac{\pi R^4 G}{2L} \cdot \varphi_{\max}$

(+10)

b) $P_{\max} = \overline{M_{\max} \cdot \dot{\varphi}} = M_{\max} \cdot \dot{\varphi}$ decrease of parallelity.

c) $P_{\max} = \frac{\pi R^4 G}{2L} \cdot \varphi_{\max} \cdot \dot{\varphi}_{\max} = \frac{\pi (1,5 \text{ cm})^4 (81,6 \text{ Pa})}{2 \cdot (2 \text{ m})} \cdot \left(\frac{2,5}{360} \cdot 2\pi \right) \cdot \left(120 \cdot \frac{1}{s} \right)$
 Wolfram Alpha for calculation
 $= 16900 \text{ W}$ (+)

(+8)

10.4 a) Equation of movement:

$M = M_R + M_B$

$I \cdot \ddot{\varphi} = -\alpha \cdot \varphi - \beta \cdot \dot{\varphi}$ (+ all M vectors point in z direction)

$\Rightarrow \ddot{\varphi} + \frac{\beta}{I} \dot{\varphi} + \frac{\alpha}{I} \varphi = 0$ with $I = \frac{1}{2} m r^2$ (+)

Solution of differential equation $\ddot{z} + 2\gamma \dot{z} + \omega_0^2 z = 0$:

$z(t) = e^{-\gamma t} (c_1 e^{i\omega_2 t} + c_2 e^{-i\omega_2 t})$ overdamped case only!!

With $\gamma = \frac{\beta}{2I}$ and $\omega_0^2 = \frac{\alpha}{I}$

($I = \frac{1}{2} m r^2$, so $\gamma = \frac{\beta}{m r^2}$)

b) For critical damping, $\gamma = \omega_0$, so $\frac{\beta}{m r^2} = \sqrt{\frac{2\alpha}{m r^2}}$

$\Rightarrow \beta = \sqrt{2\alpha m r^2}$

{numbers
 $= 0,0158 \frac{\text{kg} \cdot \text{m}^2}{\text{s}}$ (+)

c) For critical damping, we have the general form

$\varphi(t) = (a + bt) e^{-\gamma t}$ (+

$\dot{\varphi}(t) = -a\gamma e^{-\gamma t} + b e^{-\gamma t} - b\gamma t e^{-\gamma t}$ $\gamma z?$

Plugging in initial conditions:

$\varphi(t=0) = a \stackrel{!}{=} \varphi_0 \Rightarrow a := \varphi_0$

$\dot{\varphi}(t=0) = -a\gamma + b \stackrel{!}{=} 0 \Rightarrow b := a\gamma = \varphi_0 \cdot \frac{\beta}{m r^2}$

(+8)

$\Rightarrow \varphi(t) = \varphi_0 (1 + \gamma t) e^{-\gamma t} = \varphi_0 \left(1 + \frac{\beta}{m r^2} t \right) \cdot e^{-\frac{\beta t}{m r^2}}$

d) Sketch: see extra sheet

e) $\varphi_0 (1 + \gamma t) e^{-\gamma t} = \varphi_0 \cdot 10^{-3} \Rightarrow$ This can't be solved analytically; numeric solution (Wolfram Alpha): $t \approx 73$ (+)

